Amendments to the Specification:

Please delete the sub-heading before paragraph [0001] and add the following new sub-headings:

--**BACKGROUND OF THE INVENTION**

1. Field of the Invention --

Please add before paragraph [0002] the following new sub-heading:

-- 2. Description of the Related Art --

Please add before paragraph [0003] the following new sub-heading:

--**SUMMARY OF THE INVENTION**--

Please delete paragraph [0004] in entirety.

Please replace paragraph [0005] with the following amended paragraph:

[0005] ~~The~~ According to the invention, the valve is designed as a hydraulically controlled hold-open valve; as a result, costs can be reduced because a hold-open valve is less expensive than the previously used solenoid valve. Solenoid valves must be purchased additionally, whereas hydraulically controlled hold-open valves can be self-fabricated. In addition, the advantage of improved fire safety is also present, because, if there should be a power failure, the valve according to the invention automatically closes the door connected to the

inventive servo door drive. No failure is possible in a case such as this, and the efficiency of the door drive is not affected. The design according to the invention also provides for the integration of overload protection into the servo door drive as well as a closing sequence control function for doors with two wings. The hydraulically controlled hold-open function makes it possible for the door to be kept open continuously, because only a low control pressure is required. A motor-pump unit, for example, can maintain such a low pressure over a very long time without the danger of overheating and with minimal burning of the brushes. In addition, the design according to the invention also provides the advantage that, after the motor has been turned off, the inventive servo door drive continues to operate without being affected by the hydraulic drive, which means that there is no need for an additional switching valve.

Please delete paragraph [0020] in entirety.

Please add before paragraph [0031] the following new sub-heading:

--**BRIEF DESCRIPTION OF THE DRAWINGS**--

Please replace paragraph [0034] with the following amended paragraph:

[0034] The nonreturn valve 15 and the relief valve 12 are installed in the sealing disk 10. The relief valve 12 is spring-loaded. In addition, the valves 12 and 15 are installed in anti-parallel fashion. The same number and type of valves ~~13-16~~ 13 and 16 are also present in the sealing disk 11, so that excess pressure can escape from the spring space into the piston 2.

Please replace paragraph [0037] with the following amended paragraph:

[0037] a 2/2-way directional control valve 21, which prevents hydraulic fluid from returning from the piston space to a tank space 8 installed in the hydraulic circuit;

Please replace paragraph [0045] with the following amended paragraph:

[0045] There is no longer a nonreturn valve in the sealing disk 11; instead, there is a direct connection 48 from the piston space 1 to the interior of the piston 2. ~~In addition, the~~ The nonreturn valve 15 and the relief valve 12 in the sealing disk 10 are ~~now~~ again connected in anti-parallel fashion.

Please replace paragraph [0048] with the following amended paragraph:

[0048] Figure 4 shows a second alternative embodiment of the hold-open valve 20. The connections 41, 42 are the same as those explained in conjunction with Figure 3. The difference with respect to the embodiment according to Figure 3 is that the nonreturn valve 23 with the spring element 26 is integrated not into the control piston 22 but rather into a bypass 50, which connects the connection of the hydraulic line 42 leading to the piston space 1 to the hydraulic line 41 leading to the pump 6. There is no through-hole inside the control piston 22. In the alternative embodiment according to Figure 4, furthermore, the closing function of the 2/2-way directional control valve is realized by a separate closing body 9 in the form of a ball. This offers the advantage that the closing body 9 centers itself inside a ~~spherical recess 53~~ conical recess 58 and positional tolerances between the control piston 22 and the valve seat are therefore

compensated. The control piston 22 is also pretensioned by the spring element 27 toward the closing body 9.

Please replace paragraph [0050] with the following amended paragraph:

[0050] Figure 6 shows a fourth alternative embodiment of the hold-open valve 20. In contrast to the previously described embodiments, the 2/2-way directional control valve is designed as a slide valve. The sliding body of the 2/2-way directional control valve is formed by a cylindrical body, which, depending on the position to which it is shifted, either closes or opens the hydraulic line 46. The hydraulic line 46 ends again at the space holding the control piston 22 and simultaneously at a space ~~52~~ 57 upstream of the hydraulic line 42, in which space a piston ~~51~~ 56 is installed. The piston 51 is free-floating and has a diameter which is much smaller than that of the control piston 22.

Please replace paragraph [0051] with the following amended paragraph:

[0051] Figure 7 shows a fifth alternative embodiment of the hold-open valve 20, in which an adjustable valve 28 is also installed between the hydraulic lines 41 and 46 so that the pressure is equalized more quickly at the control piston 22 when the drive is shut off. This has the effect of increasing the switching speed of the 2/2-way directional control valve 21. At the same time, the additional valve 28 can be adjusted in such a way as to limit to an acceptable degree the additional leakage which occurs during operation as a result of the presence of the valve.